

## Compressible Flow

## Mach Number:

## $M=\underline{V}$ <br> C

$c=\sqrt{k R T} ; \quad \mathrm{R}=$ Ideal gas constant (for air, $\mathrm{R}=287 \mathrm{~N} . \mathrm{m} / \mathrm{kgK}$ )
$\mathrm{M}<0.3$ incompressible flow
$M<\mathbf{1 . 0}$ subsonic flow
M@1.0 transonic flow
$M=\mathbf{1 . 0}$ sonic flow
$M>1.0$ supersonic flow
M > 3.0 hypersonic flow


## Class 18: Compressible Flow - Ideal gas law

## Ideal/Perfect Gas Law (equation of state for an ideal gas):

Changes in gas density directly related to changes in pressure and temperature through the equation

$$
p=\rho R T
$$

,$p$ is the pressure, $\rho$ is the density, $T$ is the absolute temperature and $R$ is a gas constant.

Mass flow rate:

$$
\dot{m}=\rho A V=\frac{P}{R T} A V
$$



$$
\Rightarrow \dot{m}=\frac{P}{R T} A * M * C=\frac{P}{R T} A * M * \sqrt{k R T}=P * A * M * \sqrt{\frac{k}{R T}}
$$

## Class 18: Compressible Flow - Example

Example: Air at Mach 1.25 passes through a circular channel 10 cm in diameter. The static pressure and temperature are 100 kpa and $30^{\circ} \mathrm{C}$ respectively. Find the mass flow rate.

Solution: The mass flow rate can be calculated as
$\dot{m}=\rho A V=\frac{P}{R T} A V=P * A * M * \sqrt{\frac{k}{R T}}$
$\Rightarrow \dot{m}=100 \mathrm{kPka} \times\left(\frac{\left(10^{3} \mathrm{~N} / \mathrm{m}^{3}\right)}{1 \mathrm{kPa}}\right) \times \frac{\pi}{4}(0.01 \mathrm{~m})^{2} \times 1.25 \sqrt{\frac{1.4}{\left(287 \mathrm{~N} . \mathrm{m} / \mathrm{kg}^{o} \mathrm{~K}\right)(273+30)^{\circ} \mathrm{K}}}$
$\Rightarrow \dot{m}=3.93 \frac{\mathrm{~kg}}{\mathrm{~s}}$

## Class 18: Compressible Flow - different processes

## - Adiabatic Process

An adiabatic process is one in which no heat is gained or lost by the system. The first law of thermodynamics with $Q=0$ shows that all the change in internal energy is in the form of work done. This puts a constraint on the heat engine process leading to the adiabatic condition. This condition can be used to derive the expression for the work done during an adiabatic process.
$P V^{k}=$ constant $=C ; \mathrm{k}=\mathrm{C}_{\mathrm{p}} / \mathrm{C}_{\mathrm{V}}$ ratio of specific heats.

- Reversible process

A reversible process is a process that, after it has taken place, can be reversed and causes no change in either the system or its surroundings.

- Isentropic flow

An isentropic flow is a flow that is both adiabatic and reversible. That is, no energy is added to the flow, and no energy losses occur due to friction or dissipative effects.

## Class 18: Compressible Flow - different processes

Relationship between temperature, density and pressure for the isentropic flow of an ideal gas:

$$
\begin{aligned}
& \left(\frac{T}{T_{0}}\right)^{k / k-1}=\left(\frac{\rho}{\rho_{0}}\right)^{k}=\left(\frac{p}{p_{0}}\right) \\
& \Rightarrow \frac{p}{\rho^{k}}=\text { constant }
\end{aligned}
$$

For isentropic flow:

$$
c=\sqrt{\left(\frac{d p}{d \rho}\right)_{s}}=\text { speed of sound }
$$

## Class 18: Compressible Flow $\longrightarrow$ Converging-Diverging Nozzle

Experiment: Converging-diverging nozzle


Subsonic $\longrightarrow$ Sonic $\longrightarrow$ Supersonic Flow

## Class 18: Effect of Variations in Flow Cross-sectional Area

Newtons $2^{\text {nd }}$ law applied to the inviscid and steady flow (Bernoulli):

$$
\begin{aligned}
& d p+1 / 2 \rho d\left(V^{2}\right)+\gamma d z=0 \\
& \Rightarrow d p+1 / 2 \rho d\left(V^{2}\right)=0 ; \text { for ideal gas P. E. term dropped }
\end{aligned}
$$

$$
\Rightarrow \frac{d p}{\rho V^{2}}=-\frac{d V}{V}
$$

$$
\dot{m}=\rho A V=\text { constant }
$$

$$
\begin{aligned}
& c=\sqrt{\left(\frac{d p}{d \rho}\right)} ; \mathbf{M}=\frac{\mathrm{V}}{\mathrm{c}} \\
& c^{2}=\left(\frac{d p}{d \rho}\right)=\frac{\mathrm{V}^{2}}{M^{2}} ; \text { or }
\end{aligned}
$$

$$
\Rightarrow \ln \rho+\ln \mathrm{A}+\ln \mathrm{V}=0
$$

$$
\Rightarrow \frac{d \rho}{\rho}+\frac{d A}{A}+\frac{d V}{V}=0
$$

$$
\Rightarrow-\frac{d V}{V}=\frac{d \rho}{\rho}+\frac{d A}{A}
$$

$$
\Rightarrow \frac{d p}{\rho V^{2}}\left(1-\frac{V^{2}}{d p / d \rho}\right)=\frac{d A}{A}
$$

## Class 18: Effect of Variations in Flow Cross-sectional Area ..cont

$$
\frac{d p}{\rho V^{2}}\left(1-M^{2}\right)=\frac{d A}{A}
$$

$$
\Rightarrow d p\left(1-M^{2}\right)=\rho V^{2} \frac{d A}{A} ; b u t
$$

$\frac{d p}{\rho V^{2}}=-\frac{d V}{V} ; s o$

$$
\frac{d V}{V}=-\frac{d A}{A} \frac{1}{\left(1-M^{2}\right)}
$$



> Subsonic flow $\begin{gathered}(\mathrm{Ma}<1) \\ d A>0 \\ d V<0\end{gathered}$
(a)


Supersonic flow ( $\mathrm{Ma}>1$ ) $d A>0$
$d V>0$

$$
\begin{aligned}
& d A<0 \\
& d V<0
\end{aligned}
$$

(b)

## Class 18: Effect of Variations in Flow Cross-sectional Area

How Mach number influences Temperature, Pressure and Density of the fluid?

$$
\begin{aligned}
& " t " \equiv \text { TOTAL OR STAGNATION } \\
& \frac{T_{t}}{T}=1+\frac{k-1}{2} M^{2} ; \quad k=\frac{c_{p}}{c_{v}} \\
& \frac{P_{t}}{P}=\left[1+\frac{k-1}{2} M^{2}\right]^{k /(k-1)} \\
& \frac{\rho_{t}}{\rho}=\left[1+\frac{k-1}{2} M^{2}\right]^{1 /(k-1)}
\end{aligned}
$$

Temperature, Pressure and density can be tabulated for a given value of $k$ (for air, $k=1.4$ ).

## Class 18: Compressible Flow $\rightarrow$ Converging-Diverging Nozzle

## Effect of Back Pressure on Flow Pattern: Shockwave and expansion


$>$ Normal Shockwave (abrupt rise or drop of pressure)
$>$ Ideally/Perfectly Expanded
$>$ Over Expanded (pressure rises at the duct exit)
$>$ Under Expanded (pressure drops at the duct exit)
$>$ Oblique Shockwave (less abrupt rise or drop of pressure)
p


## Class 18: Compressible Flow $\rightarrow$ Converging-Diverging Nozzle

## Choked Flow/ Unchoked Flow:



Shockwave: Each abrupt pressure rise within and at the exit of the flow passage occurs across a very thin discontinuity in the flow called a Normal Shockwave.


## Compressible Flow - Table

## COMPRESSIBLE FLOW TABLES FOR

AN IDEAL GAS WITH $k=1.4$
Subsonic Flow

| M | $p / p_{t}$ | $\rho / \rho_{\mathrm{t}}$ | $T / T_{t}$ | $A / A *$ |
| :---: | :---: | :---: | :---: | :---: |
| 0.00 | 1.0000 | 1.0000 | 1.0000 | $\infty$ |
| 0.05 | 0.9983 | 0.9988 | 0.9995 | 11.5914 |
| 0.10 | 0.9930 | 0.9950 | 0.9980 | 5.8218 |
| 0.15 | 0.9844 | 0.9888 | 0.9955 | 3.9103 |
| 0.20 | 0.9725 | 0.9803 | 0.9921 | 2.9630 |
| 0.25 | 0.9575 | 0.9694 | 0.9877 | 2.4027 |
| 0.30 | 0.9395 | 0.9564 | 0.9823 | 2.0351 |
| 0.35 | 0.9188 | 0.9413 | 0.9761 | 1.7780 |
| 0.40 | 0.8956 | 0.9243 | 0.9690 | 1.5901 |
| 0.45 | 0.8703 | 0.9055 | 0.9611 | 1.4487 |
| 0.50 | 0.8430 | 0.8852 | 0.9524 | 1.3398 |
| 0.52 | 0.8317 | 0.8766 | 0.9487 | 1.3034 |
| 0.54 | 0.8201 | 0.8679 | 0.9449 | 1.2703 |
| 0.56 | 0.8082 | 0.8589 | 0.9410 | 1.2403 |
| 0.58 | 0.7962 | 0.8498 | 0.9370 | 1.2130 |
| 0.60 | 0.7840 | 0.8405 | 0.9328 | 1.1882 |
| 0.62 | 0.7716 | 0.8310 | 0.9286 | 1.1657 |
| 0.64 | 0.7591 | 0.8213 | 0.9243 | 1.1452 |
| 0.66 | 0.7465 | 0.8115 | 0.9199 | 1.1265 |
| 0.68 | 0.7338 | 0.8016 | 0.9153 | 1.1097 |
| 0.70 | 0.7209 | 0.7916 | 0.9107 | 1.0944 |
| 0.72 | 0.7080 | 0.7814 | 0.9061 | 1.0806 |
| 0.74 | 0.6951 | 0.7712 | 0.9013 | 1.0681 |
| 0.76 | 0.6821 | 0.7609 | 0.8964 | 1.0570 |
| 0.78 | 0.6691 | 0.7505 | 0.8915 | 1.0471 |
| 0.80 | 0.6560 | 0.7400 | 0.8865 | 1.0382 |
| 0.82 | 0.6430 | 0.7295 | 0.8815 | 1.0305 |
| 0.84 | 0.6300 | 0.7189 | 0.8763 | 1.0237 |
| 0.86 | 0.6170 | 0.7083 | 0.8711 | 1.0179 |
| 0.88 | 0.6041 | 0.6977 | 0.8659 | 1.0129 |
| 0.90 | 0.5913 | 0.6870 | 0.8606 | 1.0089 |
| 0.92 | 0.5785 | 0.6764 | 0.8552 | 1.0056 |
| 0.94 | 0.5658 | 0.6658 | 0.8498 | 1.0031 |
| 0.96 | 0.5532 | 0.6551 | 0.8444 | 1.0014 |
| 0.98 | 0.5407 | 0.6445 | 0.8389 | 1.0003 |
| 1.00 | 0.5283 | 0.6339 | 0.8333 | 1.0000 |

## A* is critical area for choked flow at throat.

Source: Roberson and Crowe, Engineering Fluid Mechanics, $6^{\text {th }}$ Edition, 1996, John Wiley and Sons.

## Compressible Flow - Table

COMPRESSIBLE FLOW TABLES FOR
AN IDEAL GAS WITH $k=1.4$ (CONTINUED)

| Supersonic Flow |  |  |  |  | Normal Shock Wave |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $M_{1}$ | $p / p_{t}$ | $\rho / \rho_{t}$ | $\boldsymbol{T} / \boldsymbol{T}_{\boldsymbol{t}}$ | A/A* | $M_{2}$ | $p_{2} / p_{1}$ | $\boldsymbol{T}_{2} / \boldsymbol{T}_{\boldsymbol{I}}$ | $\boldsymbol{P}_{t_{2}} / P_{t_{1}}$ |
| 1.00 | 0.5283 | 0.6339 | 0.8333 | 1.000 | 1.000 | 1.000 | 1.000 | 1.0000 |
| 1.01 | 0.5221 | 0.6287 | 0.8306 | 1.000 | 0.9901 | 1.023 | 1.007 | 0.9999 |
| 1.02 | 0.5160 | 0.6234 | 0.8278 | 1.000 | 0.9805 | 1.047 | 1.013 | 09999 |
| 1.03 | 0.5099 | 0.6181 | 0.8250 | 1.001 | 0.9712 | 1.071 | 1.020 | 0.9999 |
| 1.04 | 0.5039 | 0.6129 | 0.8222 | 1.001 | 0.9620 | 1.095 | 1.026 | 0.9999 |
| 1.05 | 0.4979 | 0.6077 | 0.8193 | 1.002 | 0.9531 | 1.120 | 1.033 | 0.9998 |
| 1.06 | 0.4919 | 0.6024 | 0.8165 | 1.003 | 0.9444 | 1.144 | 1.039 | 0.9997 |
| 1.07 | 0.4860 | 0.5972 | 0.8137 | 1.004 | 0.9360 | 1.169 | 1.046 | 0.9996 |
| 1.08 | 0.4800 | 0.5920 | 0.8108 | 1.005 | 0.9277 | 1.194 | 1.052 | 0.9994 |
| 1.09 | 0.4742 | 0.5869 | 0.8080 | 1.006 | 0.9196 | 1.219 | 1.059 | 0.9992 |
| 1.10 | 0.4684 | 0.5817 | 0.8052 | 1.008 | 0.9118 | 1.245 | 1.065 | 0.9989 |
| 1.11 | 0.4626 | 0.5766 | 0.8023 | 1.010 | 0.9041 | 1.271 | 1.071 | 0.9986 |
| 1.12 | 0.4568 | 0.5714 | 0.7994 | 1.011 | 0.8966 | 1.297 | 1.078 | 0.9982 |
| 1.13 | 0.4511 | 0.5663 | 0.7966 | 1.013 | 0.8892 | 1.323 | 1.084 | 0.9978 |
| 1.14 | 0.4455 | 0.5612 | 0.7937 | 1.015 | 0.8820 | 1.350 | 1.090 | 0.9973 |
| 1.15 | 0.4398 | 0.5562 | 0.7908 | 1.017 | 0.8750 | 1.376 | 1.097 | 0.9967 |
| 1.16 | 0.4343 | 0.5511 | 0.7879 | 1.020 | 0.8682 | 1.403 | 1.103 | 0.9961 |
| 1.17 | 0.4287 | 0.5461 | 0.7851 | 1.022 | 0.8615 | 1.430 | 1.109 | 0.9953 |
| 1.18 | 0.4232 | 0.5411 | 0.7822 | 1.025 | 0.8549 | 1.458 | 1.115 | 0.9946 |
| 1.19 | 0.4178 | 0.5361 | 0.7793 | 1.026 | 0.8485 | 1.485 | 1.122 | 0.9937 |
| 1.20 | 0.4124 | 0.5311 | 0.7764 | 1.030 | 0.8422 | 1.513 | 1.128 | 0.9928 |
| 1.21 | 0.4070 | 0.5262 | 0.7735 | 1.033 | 0.8360 | 1.541 | 1.134 | 0.9918 |
| 1.22 | 0.4017 | 0.5213 | 0.7706 | 1.037 | 0.8300 | 1.570 | 1.141 | 0.9907 |
| 1.23 | 0.3964 | 0.5164 | 0.7677 | 1.040 | 0.8241 | 1.598 | 1.147 | 0.9896 |
| 1.24 | 0.3912 | 0.5115 | 0.7648 | 1.043 | 0.8183 | 1.627 | 1.153 | 0.9884 |
| 1.25 | 0.3861 | 0.5067 | 0.7619 | 1.047 | 0.8126 | 1.656 | 1.159 | 0.9871 |
| 1.30 | 0.3609 | 0.4829 | 0.7474 | 1.066 | 0.7860 | 1.805 | 1.191 | 0.9794 |
| 1.35 | 0.3370 | 0.4598 | 0.7329 | 1.089 | 0.7618 | 1.960 | 1.223 | 0.9697 |
| 1.40 | 0.3142 | 0.4374 | 0.7184 | 1.115 | 0.7397 | 2.120 | 1.255 | 0.9582 |
| 1.45 | 0.2927 | 0.4158 | 0.7040 | 1.144 | 0.7196 | 2.286 | 1.287 | 0.9448 |
| 1.50 | 0.2724 | 0.3950 | 0.6897 | 1.176 | 0.7011 | 2.458 | 1.320 | 0.9278 |
| 1.55 | 0.2533 | 0.3750 | 0.6754 | 1.212 | 0.6841 | 2.636 | 1.354 | 0.9132 |
| 1.60 | 0.2353 | 0.3557 | 0.6614 | 1.250 | 0.6684 | 2.820 | 1.388 | 0.8952 |
| 1.65 | 0.2184 | 0.3373 | 0.6475 | 1.292 | 0.6540 | 3.010 | 1.423 | 0.8760 |
| 1.70 | 0.2026 | 0.3197 | 0.6337 | 1.338 | 0.6405 | 3.205 | 1.458 | 0.8557 |
| 1.75 | 0.1878 | 0.3029 | 0.6202 | 1.386 | 0.6281 | 3.406 | 1.495 | 0.8346 |

## Compressible Flow - Table

COMPRESSIBLE FLOW TABLES FOR
AN IDEAL GAS WITH $k=1.4$ (CONTINUED)

| Supersonic Flow |  |  |  | Normal Shock Wave |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $M_{I}$ | $p / p_{t}$ | $\rho / \rho_{\mathrm{t}}$ | T/T $\boldsymbol{T}_{\boldsymbol{t}} \quad$ A/A* | $M_{2} \quad p_{2} / p_{1}$ | $\boldsymbol{T}_{2} / \boldsymbol{T}_{1}$ | $P_{t_{2}} / P_{t_{1}}$ |
| 1.80 | 0.1740 | 0.2868 | 0.60681 .439 | $\begin{array}{lll}0.6165 & 3.613\end{array}$ | 1.532 | 0.8127 |
| 1.85 | 0.1612 | 0.2715 | 0.59361 .495 | 0.60573 .826 | 1.569 | 0.7902 |
| 1.90 | 0.1492 | 0.2570 | 0.58071 .555 | 0.59564 .045 | 1.608 | 0.7674 |
| 1.95 | 0.1381 | 0.2432 | 0.56801 .619 | 0.58624 .270 | 1.647 | 0.7442 |
| 2.00 | 0.1278 | 0.2300 | 0.55561 .688 | 0.57744 .500 | 1.688 | 0.7209 |
| 2.10 | 0.1094 | 0.2058 | 0.53131 .837 | 0.56134 .978 | 1.770 | 0.6742 |
| 2.20 | $0.9352^{-1^{*}}$ | 0.1841 | 0.50812 .005 | $0.5471 \quad 5.480$ | 1.857 | 0.6281 |
| 2.30 | $0.7997^{-1}$ | 0.1646 | 0.48592 .193 | 0.53446 .005 | 1.947 | 0.5833 |
| 2.50 | $1.5853^{-1}$ | 0.1317 | 0.44442 .637 | $0.5130 \quad 7.125$ | 2.138 | 0.4990 |
| 2.60 | $0.5012^{-1}$ | 0.1179 | $0.4252 \quad 2.896$ | 0.50397 .720 | 2.238 | 0.4601 |
| 2.70 | $0.4295^{-1}$ | 0.1056 | 0.40683 .183 | 0.49568 .338 | 2.343 | 0.4236 |
| 2.80 | $0.3685^{-1}$ | $0.9463^{-1}$ | 0.38943 .500 | 0.48828 .980 | 2.451 | 0.3895 |
| 2.90 | $0.3165^{-1}$ | $0.8489^{-1}$ | 0.37293 .850 | $0.4814 \quad 9.645$ | 2.563 | 0.3577 |
| 3.00 | $0.2722^{-1}$ | $0.7623^{-1}$ | 0.35714 .235 | 0.475210 .33 | 2.679 | 0.3283 |
| 3.50 | $0.1311^{-1}$ | $0.4523^{-1}$ | 0.28996 .790 | 0.451214 .13 | 3.315 | 0.2129 |
| 4.00 | $0.6586^{-2}$ | $0.2766^{-1}$ | 0.238110 .72 | 0.435018 .50 | 4.047 | 0.1388 |
| 4.50 | $0.3155^{-2}$ | $0.1745^{-1}$ | 0.198016 .56 | 0.423623 .46 | 4.875 | $0.9170^{-1}$ |
| 5.00 | $0.1890^{-2}$ | $0.1134^{-1}$ | 0.166725 .00 | 0.415229 .00 | 5.800 | $0.6172^{-1}$ |
| 5.50 | $0.1075^{-2}$ | $0.7578^{-2}$ | 0.141836 .87 | 0.409035 .13 | 6.822 | $0.4236^{-1}$ |
| 6.00 | $0.6334^{-2}$ | $0.5194^{-2}$ | 0.122053 .18 | 0.404241 .83 | 7.941 | $0.2965^{-1}$ |
| 6.50 | $0.3855^{-2}$ | $0.3643^{-2}$ | 0.105875 .13 | 0.400449 .13 | 9.156 | $0.2115^{-1}$ |
| 7.00 | $0.2416^{-3}$ | $0.2609^{-2}$ | $0.9259^{-1} 104.1$ | 0.397457 .00 | 10.47 | $0.1535^{-1}$ |
| 7.50 | $0.1554^{-3}$ | $0.1904^{-2}$ | $0.8163^{-1} 141.8$ | 0.394965 .46 | 11.88 | $0.1133{ }^{-1}$ |
| 8.00 | $0.1024^{-3}$ | $0.1414^{-2}$ | $0.7246^{-1} 190.1$ | 0.392974 .50 | 13.39 | $0.8488^{-2}$ |
| 8.50 | $0.6898^{-4}$ | $0.1066^{-3}$ | $0.6472^{-1} 251.1$ | 0.391284 .13 | 14.99 | $0.6449^{-2}$ |
| 9.00 | $0.4739^{-4}$ | $0.8150^{-3}$ | $0.5814^{-1} 327.2$ | 0.389894 .33 | 16.69 | $0.4964{ }^{-2}$ |
| 9.50 | $0.3314^{-4}$ | $0.6313^{-3}$ | $0.5249^{-1} 421.1$ | 0.3886105 .1 | 18.49 | $0.3866^{-2}$ |
| 10.00 | $0.2356^{-4}$ | $0.4948^{-3}$ | $0.4762^{-1} 535.9$ | 0.3876116 .5 | 20.39 | $0.3045^{-2}$ |

## Compressible Flow - Table

Properties of the U.S. Standard Atmosphere (SI Units)

| Acceleration of <br> Gravity, |  |  |  |  |  |  |  |  |  |  |  |
| ---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Altitude <br> $(\mathrm{m})$ |  |  |  |  |  |  | Temperature <br> $\left(\mathrm{C}^{\circ}\right)$ | $g$ <br> $\left(\mathrm{~m} / \mathrm{s}^{2}\right)$ | $p$ <br> $[\mathrm{~Pa}, \mathrm{abs}]$ | Density, <br> $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ | Dynamic <br> Viscosity, <br> $\mu$ |
| 1,000 | 21.50 | 9.810 | $1.139 \mathrm{E}+5$ | $1.347 \mathrm{E}+0$ | $1.821 \mathrm{E}-5$ |  |  |  |  |  |  |
| 0 | 15.00 | 9.807 | $1.013 \mathrm{E}+5$ | $1.225 \mathrm{E}+0$ | $1.789 \mathrm{E}-5$ |  |  |  |  |  |  |
| 1,000 | 8.50 | 9.804 | $8.988 \mathrm{E}+4$ | $1.112 \mathrm{E}+0$ | $1.758 \mathrm{E}-5$ |  |  |  |  |  |  |
| 2,000 | 2.00 | 9.801 | $7.950 \mathrm{E}+4$ | $1.007 \mathrm{E}+0$ | $1.726 \mathrm{E}-5$ |  |  |  |  |  |  |
| 3,000 | -4.49 | 9.797 | $7.012 \mathrm{E}+4$ | $9.093 \mathrm{E}-1$ | $1.694 \mathrm{E}-5$ |  |  |  |  |  |  |
| 4,000 | -10.98 | 9.794 | $6.166 \mathrm{E}+4$ | $8.194 \mathrm{E}-1$ | $1.661 \mathrm{E}-5$ |  |  |  |  |  |  |
| 5,000 | -17.47 | 9.791 | $5.405 \mathrm{E}+4$ | $7.364 \mathrm{E}-1$ | $1.628 \mathrm{E}-5$ |  |  |  |  |  |  |
| 6,000 | -23.96 | 9.788 | $4.722 \mathrm{E}+4$ | $6.601 \mathrm{E}-1$ | $1.595 \mathrm{E}-5$ |  |  |  |  |  |  |
| 7,000 | -30.45 | 9.785 | $4.111 \mathrm{E}+4$ | $5.900 \mathrm{E}-1$ | $1.561 \mathrm{E}-5$ |  |  |  |  |  |  |
| 8,000 | -36.94 | 9.782 | $3.565 \mathrm{E}+4$ | $5.258 \mathrm{E}-1$ | $1.527 \mathrm{E}-5$ |  |  |  |  |  |  |
| 9,000 | -43.42 | 9.779 | $3.080 \mathrm{E}+4$ | $4.671 \mathrm{E}-1$ | $1.493 \mathrm{E}-5$ |  |  |  |  |  |  |
| 10,000 | -49.90 | 9.776 | $2.650 \mathrm{E}+4$ | $4.135 \mathrm{E}-1$ | $1.458 \mathrm{E}-5$ |  |  |  |  |  |  |
| 15,000 | -56.50 | 9.761 | $1.211 \mathrm{E}+4$ | $1.948 \mathrm{E}-1$ | $1.422 \mathrm{E}-5$ |  |  |  |  |  |  |

## Class 18: Compressible Flow

Problem: For an aircraft flying at Mach 3.0 at an altitude of $10,000 \mathrm{~m}\left(\mathrm{~T}=-50^{\circ} \mathrm{C}\right)$, estimate the surface temperature at the nose.


Solution: From the table, @ $\mathbf{M}=\mathbf{3}$,

$$
\begin{aligned}
& \frac{T}{T_{t}}=0.3571 \\
& \Rightarrow \frac{T_{t}}{T}=2.80 \\
& \Rightarrow T_{t}=2.8(273-50)=624^{\circ} \mathrm{K}=351^{\circ} \mathrm{C}
\end{aligned}
$$

## Class 18: Compressible Flow

Problem: A converging nozzle has an exit area of $500 \mathrm{~mm}^{2}$. Air enters this nozzle from a reservoir at $1000 \mathrm{kpa} \& 360 \mathrm{~K}$. The exit pressure is 800 kPa . Find the mass flow rate through the nozzle.
Solution: First we determine the pressure ratio,

$$
\frac{P_{E}}{P_{t}}=\frac{800}{1000}=0.8(\text { subsonic }>0.5283)
$$

Using this value, we get from the table

$$
\begin{aligned}
& M_{E}=0.573 \\
& \Rightarrow \frac{T_{E}}{T_{t}}=0.9381 \\
& \Rightarrow T_{E}=0.9381 \times 360=337.7^{\circ} \mathrm{K}
\end{aligned}
$$



Now
$c_{E}=\sqrt{k R T_{E}}=\sqrt{(1.4)(287)(337.7)}=368.4 \mathrm{~m} / \mathrm{s}$
Hence

$$
\begin{aligned}
& V_{E}=M_{E} \cdot c_{E}=(0.573) \cdot(368.4 \mathrm{~m} / \mathrm{s})=211.1 \mathrm{~m} / \mathrm{s} \\
& \rho_{E}=\frac{P_{E}}{R T_{E}}=\frac{800 \times 10^{3} \mathrm{~N} / \mathrm{m}^{2}}{\left(287 \mathrm{~J} / \mathrm{kg} .^{o} \mathrm{~K}\right)\left(337.7^{o} \mathrm{~K}\right)}=8.254 \mathrm{~kg} / \mathrm{m}^{3} \\
& \dot{m}=\rho_{E} V_{E} A_{E}=\left(8.254 \mathrm{~kg} / \mathrm{m}^{3}\right)(211.1 \mathrm{~m} / \mathrm{s})\left(500 \times 10^{-6} \mathrm{~m}^{2}\right) \\
& \Rightarrow \dot{m}=0.871 \mathrm{~kg} / \mathrm{s}
\end{aligned}
$$

## Class 18: Compressible Flow

Problem: The stagnation pressure indicated by a Pitot tube mounted on an airplane in flight is 45 kPa (abs). If the aircraft is cruising in standard atmosphere at an altitude of $10,000 \mathrm{~m}$, determine the speed and Mach number involved.

Solution: First we find the pressure ratio
$P=2.65 \times 10^{4} \mathrm{~Pa}, \quad P_{t}=45 \times 10^{3} \mathrm{~Pa}$
$\Rightarrow \frac{P}{P_{t}}=0.589($ subsonic $>0.5283)$
Now from the table, we get
$M \cong 0.90$
Therefore,

$V=M \cdot c=M \sqrt{k R T}$
$\Rightarrow V=0.90 \sqrt{(1.4)(287)(273-49.90)}=269 \mathrm{~m} / \mathrm{s}$

