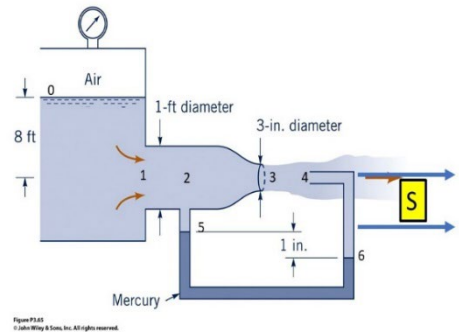


BERNOULLI & UNITS STUDY AID

1. Pressurized water flows steadily from a large, closed tank. Follow the path¹ and determine:
 - a. Pressure difference (lbf/ft²)
 - b. Volume Flow Rate (ft³/s)
 - c. Pressure in the space above the surface of the water in the tank (lbf/ft²)
 - d. Pressure at the inlet of the 1" ft Diameter section (lbf/ft²)

$$Sg_{HG} = 13.55$$



FIRST STEP: LABEL ALL IMPORTANT POINTS

- **Static Pressure Taps**
- **Stagnation Pressure Taps**
- **Free Surfaces**
- **Inlet/Outlets/Free Jets**
- **Change in Diameters**
- **Manometer Fluid Interfaces**
- **Unknown Dimensions**

Next Step: Apply Manometry "IF" Defined
 NEED PRESSURE DIFFERENCE
 BERRY POINT-to-POINT
 Use **DUCT** Points Aligned With Static/Stagnation **TAPS**

$$P_2 + \Delta P_{2-4} = P_4$$

$$P_2 + \cancel{\gamma_{H20} S} + \gamma_{Hg} \left(\frac{1}{12}\right) - \gamma_{H20} \left(\frac{1}{12} + \cancel{S}\right) = P_4$$

$$P_2 - P_4 = \frac{1}{12} (\gamma_{H20} - \gamma_{Hg})$$

$$\frac{P_2 - P_4}{\gamma_{H20}} = \frac{1}{12} (1 - Sg_{HG}) = -1.0458 \text{ ft}$$

NO OTHER APPROACH IS ACCEPTABLE

¹ NO NEED TO MOVE FORWARD IF PATH IS NOT FOLLOWED

NEXT STEP:

APPLY BERNOULLI: **SAME POINTS**

$$\dot{m}_2 \left(\frac{P_2}{\gamma_{H20}} + \frac{z_2 = z_4}{z_2} + \frac{V_2^2}{2g} \right) = \dot{m}_4 \left(\frac{P_4}{\gamma_{H20}} + \frac{z_2 = z_4}{z_4} + \frac{V_4^2}{2g} \right) \quad \text{=0:Stagnation}$$

One Inlet/One Exit

$$\dot{m}_2 = \dot{m}_4$$

$$\frac{P_2 - P_4}{\gamma_{H20}} = -\frac{V_2^2}{2g}$$

COMBINE WITH MANOMETRY

$$\frac{P_2 - P_4}{\gamma_{H20}} = \frac{1}{12}(1 - Sg_{Hg}) = -1.0458 ft$$

$$\frac{P_2 - P_4}{\gamma_{H20}} = -\frac{V_2^2}{2g} = -1.0458 ft$$

SOLVE

$$V_2 = \sqrt{(1.0458 ft) 2 \cdot 32.2 ft/s^2} = 8.21 ft/s$$

$$Q = V_2 A_2$$

$$Q = 8.21 ft/s \cdot \pi \frac{D_2^2 [ft^2]}{4} = 6.45 \frac{ft^3}{s}$$

PRESSURE ABOVE WATER

Apply Bernoulli 0-3

$$\frac{P_0}{\gamma_{H20}} + \frac{z_0 = 8 ft}{z_0} + \frac{V_0^2}{2g} = \frac{P_3}{\gamma_{H20}} + \frac{z_3 = 0 ft}{z_3} + \frac{V_3^2}{2g}$$

$$P_0 = \left(\frac{V_3^2 \left[\frac{ft^2}{s^2} \right]}{2g \left[\frac{ft}{s^2} \right]} - 8 ft \right) \gamma_{H20} \left[\frac{lbf}{ft^3} \right] =$$

$$P_0 = 16230.12 \frac{lbf}{ft^2} \left[\frac{1 ft^2}{144 in^2} \right] = 112.7 psig$$

$$V_3 = \frac{Q}{A_3} = \frac{6.45 \frac{ft^3}{s}}{0.04909 ft^2} = 131.4 ft/s$$

PRESSURE AT INLET

Apply Bernoulli 0-1

$$\frac{P_0}{\gamma_{H20}} + \frac{z_0 = 8 ft}{z_0} + \frac{V_0^2}{2g} = \frac{P_1}{\gamma_{H20}} + \frac{z_1 = 0 ft}{z_1} + \frac{V_1^2}{2g}$$

$$\frac{P_1}{\gamma_{H20}} = \frac{P_0}{\gamma_{H20}} + z_0 - \frac{V_1^2}{2g}$$

$$P_1 \left[\frac{lbf}{ft^2} \right] = \left[\frac{P_0}{\gamma_{H20}} [ft] + z_0 [ft] - \frac{V_1^2}{2g} [ft] \right] \gamma_{H20} \left[\frac{lbf}{ft^3} \right]$$

2. The temperature distribution in a fluid is given below along with its velocity vector field:

$$T[C](x, y, t) = \left(10[x]^{\frac{1}{3}} + 5[ty]^{\frac{-2}{7}} + 24[e^{-\alpha t}] \right) [C]$$

"10"

$$[C] = [m]^{\frac{1}{3}} \rightarrow [C] = \left[\frac{C}{m^{\frac{1}{3}}} \right]$$

"5"

$$[C] = [s][m]^{-2/7} \rightarrow [C] = \frac{C}{s - m^{-2/7}} = \left[\frac{Cm^{2/7}}{s} \right]$$

"24"

$$[C] = [1] \rightarrow [C] = C$$

" α "

$$[1] = [s] \rightarrow [C] = \frac{1}{s}$$

$$\vec{V} \left[\frac{m}{s} \right] (x, y, t) = u\hat{i} + v\hat{j}$$

$$= \left((20[xt]^{\frac{-3}{14}} + 45[y]^2)\hat{i} - (23[x]^2 + 50[x^{\frac{6}{7}}yt])\hat{j} \right) \left[\frac{m}{s} \right]$$

χ Through this experience, I began to understand the fundamental concepts of the course, thanks to your focus upon the units of each problem. It sounds like a little thing, but is one that is a very important detail to double check the math through each problem. At times I was frustrated and felt as though I could not understand the material, but with persistence and trying to understand the big picture, I began to make much progress. I will remember this class for how it showed me I can learn a great deal in a short amount of time. I appreciate you providing the external pressure to unlock the better engineer in all of us students.ψ

Former Student, Spring 2022

$$\vec{V} \left[\frac{m}{s} \right] (x, y, t) = u\hat{i} + v\hat{j}$$

$$= \left((20[x]t^{\frac{-3}{14}} + 45[y^2])\hat{i} - (23[x^2 + 50[x^{\frac{6}{7}}yt])\hat{j} \right) \left[\frac{m}{s} \right]$$

"20"

$$\frac{m}{s} = [] m s^{\frac{-3}{14}} \rightarrow [] = \frac{\frac{m}{s}}{m s^{\frac{-3}{14}}} \rightarrow s^{-1} s^{\frac{3}{14}} \rightarrow \left[\frac{-11}{s^{\frac{11}{14}}} \right]$$

"45"

$$\frac{m}{s} = [] m^2 \rightarrow [] = \frac{\frac{m}{s}}{m^2} = \left[\frac{1}{m-s} \right]$$

"23"

$$\frac{m}{s} = [] m^2 \rightarrow [] = \frac{\frac{m}{s}}{m^2} = \left[\frac{1}{m-s} \right]$$

"50"

$$\frac{m}{s} = [] m^{\frac{6}{7}} m s \rightarrow [] = \frac{\frac{m}{s}}{m^{\frac{6}{7}} m s} = \frac{m}{m^{\frac{13}{7}} s^2} = \left[\frac{1}{m^{\frac{6}{7}} s^2} \right]$$

where t is in seconds, x and y are in meters, and T is in degrees C. **FOLLOW THE PATH AND:**

- Determine the correct units for the brackets []?
- Using the definition of the TOTAL, or MATERIAL DERIVATIVE, determine the "parametric expression" for the time rate of change of temperature $\left(i.e., \frac{DT}{Dt} \left[\frac{C}{s} \right] \right)$ of a fluid traveling with velocity field above $(i.e., \vec{V} \left[\frac{m}{s} \right] (x, y, t))$, **AND**, verify units:
- Determine the spatial derivatives **AND** verify units: $\frac{\partial T}{\partial y} \left[\frac{C}{m} \right]$ and $\frac{\partial u}{\partial x} \left[\frac{m/s}{m} \right]$.

$$T[C](x, y, t) = \left(10[x]^{\frac{1}{3}} + 5[t]y^{\frac{-2}{7}} + 24[e^{-\alpha t}] \right) [C]$$

$$\frac{DT[C]}{Dt [s]} = \frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y}$$

$$= \underbrace{-\alpha \left[\frac{1}{s} \right] 24[C] e^{-\alpha t}}_{\frac{\partial T}{\partial t}} \left\{ \frac{[C]}{[s]} \right\} + \left\{ u \left[\frac{m}{s} \right] 10 \left[\frac{C}{m^{\frac{1}{3}}} \right] \frac{1}{3} x^{\frac{-2}{3}} \left[m^{\frac{-2}{3}} \right] \right\}_{\frac{\partial T}{\partial x}} \left\{ \frac{[C]}{[s]} \right\} + \left\{ v \left[\frac{m}{s} \right] 5 \left[\frac{C - m^{2/7}}{s} \right] (-2/7) t [\cancel{s}] y^{\frac{-9}{7}} [m^{\frac{9}{7}}] \right\}_{\frac{\partial T}{\partial y}} \left\{ \frac{[C]}{[s]} \right\}$$

“Fluid Mechanics has helped me build a stronger problem-solving skill by helping me not focus on the problem itself, but on the path that must be followed.”

Former Student, Winter 2022

Former Student, Winter 2022

$$\frac{\partial T}{\partial x} \left[\frac{C}{m} \right], \frac{\partial T}{\partial y} \left[\frac{C}{m} \right], \text{ and } \frac{\partial u}{\partial x} \left[\frac{m/s}{m} \right]$$

$$T[C](x, y, t) = \left(10[x]^{\frac{1}{3}} + 5[t]y^{\frac{-2}{7}} + 24[e^{-\alpha t}] \right) [C]$$

$$\frac{\partial T}{\partial x} \left[\frac{C}{m} \right] = 10 \left[\frac{C}{m^{\frac{1}{3}}} \right] \frac{1}{3} x^{\frac{-2}{3}} \left[m^{\frac{-2}{3}} \right] \rightarrow [C] \left[m^{\frac{\frac{3}{3}-1}{3} \frac{2-1}{3}} \right] \rightarrow \frac{[C]}{[m]}$$

$$\frac{\partial T}{\partial y} \left[\frac{C}{m} \right] = -\frac{10}{7} \left[\frac{Cm^{2/7}}{s} \right] t [\cancel{s}] y^{\frac{-9}{7}} [m^{\frac{-9}{7}}] \rightarrow [C] \left[m^{\frac{2-9}{7}} \right] \rightarrow \frac{[C]}{[m]}$$

$$u = 20 \left[\frac{-11}{s^{14}} \right] x t^{\frac{-3}{14}} + 45 [y]^2$$

$$\frac{\partial u}{\partial x} \left[\frac{m/s}{m} \right] = \frac{-3}{14} 20 \left[\frac{-11}{s^{14}} \right] t^{\frac{-3}{14}} \left[\frac{-3}{s^{14}} \right] \rightarrow \left[\frac{1}{s} \right]$$